

ACTIVE ENGINE MOUNTING SYSTEM BASED ON NEURAL NETWORK CONTROL

 $\mathbf{B}\mathbf{Y}$

FADLY JASHI DARSIVAN

A thesis submitted in fulfillment of the requirement for the degree of Doctor of Philosophy

> Kulliyyah of Engineering International Islamic University Malaysia

> > August 2010

ABSTRACT

In the automotive industry some components and subassemblies which were initially made of steel are now being replaced with alloys and composites which have a higher strength to weight ratio. Therefore, today's vehicles are lighter, stronger and thus have small fuel consumption. However, mounting a more powerful engine to a lighter vehicle could cause vibration induced by the dynamics of the engine and thus affecting the comfort of the passenger. One way to overcome this predicament is to modify the mounting of the engine by introducing an active engine mounting (AEM) system which consists of passive rubber mount and a linear force actuator. At the correct frequency the linear force actuator would trigger a force which has a magnitude approximately equal to the engine's disturbance force but opposite in direction. With this the force transmitted to the chassis of the vehicle would then be minimized and increases passenger's comfort. In controlling the system, especially the force actuator, numerous controllers have been introduced which include but not limited to H₂ controller, hybrid of feedback and feedforward, filtered X-LMS controller, optimal controller based on Haar wavelet and other classical feedback and feedforwad controllers. Determining the controller parameters could be a major and difficult task to perform since these parameters are based on the mathematical model of the engine-chassis system which also includes the mathematical model of the engine disturbance. In this thesis an intelligent controller namely the neural network controller has been introduced to reduce controller parameters identification. The system considered in this research includes two degree and multi degree of freedom systems. The dynamics of a nonlinear actuator was also included. Two types of neural network controller that has been used in this research namely the nonlinear auto regressive moving average (NARMA-L2) and the extended minimal resource allocating network (EMRAN). The performance of the neural network based controllers was then compared with classical controller such as PID for two degree of freedom system and a Linear Ouadratic Regulator (LOR) controller for the multi degree of freedom system. The ability of the EMRAN to be trained online makes it advantageous for a non-model based controller. The EMRAN neural network has the ability to add and prune hidden layer neurons and for the purpose of efficiency and additional advantage was the adoption of the "winner-takes-all" algorithm. Results show that the EMRAN controller perform much better as compared to PID and LQR controllers for the purpose of active vibration isolation based on the reduction of the force transmitted to the chassis of the vehicle.

ملخص البحث

اليوم المركبات هي أخف وزنا وأكثر قوة ، وبالتالي يكون صغيرة من استهلاك الوقود. ومع ذلك ، من تصاعد أقوى محرك للسيارة أخف ويمكن أن تسبب له تأثير الاهتزازات الناجمة عن ديناميات للمحرك ، وبالتالي تؤثر على الراحة للركاب. طريقة واحدة للتغلب على هذا المأزق هو تعديل المتزايدة للمحرك عن طريق إدخال محركا نشطا متصاعدة (AEM) ، وهو النظام الذي يتكون من المطاط السلبي وخطى في وتيرة تصحيح خطية قوة المحرك من شأنه أن يؤدي إلى القوة التي يبلغ حجمها مساويا تقريبا لقوة المحرك والاضطرابات ولكن في الاتجاه المعاكس. مع هذه القوة والتي أحيلت إلى هيكل السيارة ثم سيكون أدنى حد ممكن ، ويزيد من راحة الركاب. في السيطرة على النظام وخاصبة صمام القوة ، والعديد من وحدات التحكم وأدخلت والتي تشمل على سبيل المثال لا الحصر تحكم 2H ، وردود فعل مختلطة من feedforward وتصفيتها إكس إم إس تحكم ، التحكم الأمثل على أساس HAAR المويجات وغيرها من ردود الفعل الكلاسيكية وتحكم feedforwad. تحديد معايير تحكم يمكن أن تكون مهمة كبيرة وصعبة لأداء هذه المعلمات منذ تستند إلى نموذج رياضى للمحرك نظام الشاسيه الذي يشمل أيضا نموذج رياضى لمحرك الاضطرابات. في هذه الأطروحة جهاز تحكم ذكى وهي الشبكة العصبية تحكم عرضه للحد من تحكم معايير تحديد الهوية. نظام النظر في هذا البحث يشمل اثنين من درجة درجة وتعدد النظم الحرية. ديناميات صمام اخطى أدرج أيضا. نوعين من تحكم الشبكة العصبية تم استخدامه في هذا البحث وهي غير الخطية تراجعية المتوسط المتحرك (NARMA- 2L) والحد الأدنى من تخصيص الموارد تمديد شبكة (EMRAN). أداء وحدات تحكم الشبكة العصبية بعد ذلك بالمقارنة مع تحكم الكلاسيكية مثل بثها على التردد اثنين درجة من الحرية والنظام الخطي من الدرجة الثانية للنباتات (LQR) للتحكم في درجة متعددة من نظام الحرية. قدرة EMRAN للتدريب على الانترنت يجعل من المفيد بالنسبة لغير نموذجية تقوم وحدة تحكم الشبكة العصبية EMRAN لديه القدرة على إضافة طبقة الخلايا العصبية وتقليم والمخفية لهذا الغرض من الكفاءة وميزة إضافية تم اعتماد "الفائز يأخذ كل شيء" خوارزمية النتائج تظهر أن تحكم EMRAN أداء أفضل بكثير بالمقارنة مع معرف المنتج وLQR وحدات تحكم لغرض عزل الاهتزاز النشطة على أساس خفض القوة التي أحيلت إلى هيكل السيارة.

APPROVAL PAGE

The thesis of Fadly Jashi Darsivan has been approved by the following:

Wahyudi Martono Supervisor

Waleed F. Faris Co- Supervisor

Momoh-Jimoh E. Salami Internal Examiner

Hishamuddin Jamaluddin External Examiner

> Mohd Nasir Taib External Examiner

Asst. Prof. Dr. Amir Akramin Shafie Chairman

DECLARATION

I hereby declare that this dissertation is the result of my own investigations, except where otherwise stated. I also declare that it has not been previously or concurrently submitted as a whole for any other degrees at IIUM or other institutions.

Fadly Jashi Darsivan

Signature Date

INTERNATIONAL ISLAMIC UNIVERSITY MALAYSA

DECLARATION OF COPYRIGHT AND AFFRIMATION OF FAIR USE OF UNPUBLISHED RESEARCH

Copyright © 2010 by Fadly Jashi Darsivan. All rights reserved.

ACTIVE ENGINE MOUNTING SYSTEM BASED ON NEURAL NETWORK CONTROL

No part of this unpublished research may be reproduced, stored in a retrieval system, or transmitted, in any form or by any means, electronic, mechanical, photocopying, recording or otherwise without prior written permission of the copyright holder except as provided below.

- 1. Any material contained in or derived from this unpublished research may only be used by others in their writing with due acknowledgement.
- 2. IIUM or its library will have the right to make and transmit copies (print or electronic) for institutional and academic purposes.
- 3. The IIUM library will have the right to make, store in a retrieval system and supply copies of this unpublished research if requested by other universities and research libraries.

Affirmed by Fadly Jashi Darsivan.

Signature

Date

To my beloved wife, Juliyana Hanim and my daughters Sophia and Nadia

And

To my beloved father Ridhuan Siradj and mother Darlina Jaffar

ACKNOWLEDGEMENTS

First and foremost I would like to thank Allah the Most Merciful and the Most Compassionate for without Him this research would have never been realized.

I would like to thank my supervisor, Almarhum Prof. Dr. Wahyudi Martono for his scholarly guidance and effort in supporting me for without his encouragement this work would have never been completed.

I would also like to thank my co-supervisor Assoc. Prof. Dr. Waleed F. Faris for his motivation and support in maintaining my interest to this field of research. For which his assistance was truly beneficial and helpful.

Not to forget my colleagues at the Kulliyah of Engineering, IIUM especially Dr. Sany Izan Ihsan who has been motivating and offering friendly advice. Not to forget other academic and non-academic staff for their assistance.

My sincere gratitude to the International Islamic University of Malaysia who has been very supportive and patience in allowing me to complete my studies.

Of course, last but not least my heart felt gratitude to my beloved wife Juliyana Hanim Bt. Abdul Raoff for her patience and understanding. To my two daughters Sophia Fadly and Nadia Fadly who have been there during my studies and making life more interesting and fun. Not to forget my deepest gratitude to my mother, Darlina Jaafar and father, Ridhuan Siradj for their love and prayers.

TABLE OF CONTENT

Abstract in Arabic	iii
Approval Page	iv
Declration Page	v
Dedication	viii
Acknowledgements	viii
List of Tables	xii
List of Figure	xiii
List of Abbreviations and Symbols	XX
CHAPTER 1: INTRODUCTION	1
1.1 Overview	1
1.1.1 Passive Engine Mounts	2
1.1.2 Semi active engine mount	2
1.1.3 Active engine mount	
1.2 Problem statement and its Significance	
1.3 Research Philosophy	4
1.3 Research Ojectives	
1.4 Research Scope	5
1.5 Research Methodology	6
1.6 Thesis Organization	7
CHADTED 2. I ITEDATIDE SUDVEV	0
2 1 Introduction	9
2.1 Introduction	و9 0
2.2 Englite Woulding System	
//	
2.2.1 Passive Engine Mount	
2.2.1Passive Engine Mount 2.2.1.1 Elastomeric Mount 2.2.1.2 Hydraulic Mount	
2.2.1Passive Engine Mount 2.2.1.1 Elastomeric Mount 2.2.1.2 Hydraulic Mount 2.2.2 Semi Active and Active Engine Mounts	
2.2.1Passive Engine Mount 2.2.1.1 Elastomeric Mount 2.2.1.2 Hydraulic Mount 2.2.2 Semi Active and Active Engine Mounts 2.2 Controller	
2.2.1Passive Engine Mount 2.2.1.1 Elastomeric Mount 2.2.1.2 Hydraulic Mount 2.2.2 Semi Active and Active Engine Mounts 2.3 Controller 2.4 Neural Network Control	
 2.2.1Passive Engine Mount 2.2.1.1 Elastomeric Mount 2.2.1.2 Hydraulic Mount 2.2.2 Semi Active and Active Engine Mounts 2.3 Controller 2.4 Neural Network Control 2 5 Summary 	
 2.2.1Passive Engine Mount 2.2.1.1 Elastomeric Mount 2.2.1.2 Hydraulic Mount 2.2.2 Semi Active and Active Engine Mounts 2.3 Controller 2.4 Neural Network Control 2.5 Summary 	
2.2.1Passive Engine Mount 2.2.1.1 Elastomeric Mount 2.2.1.2 Hydraulic Mount 2.2.2 Semi Active and Active Engine Mounts 2.3 Controller 2.4 Neural Network Control 2.5 Summary CHAPTER 3: MODELING OF THE ENGINE MOUNTING SYSTEM	
 2.2.1Passive Engine Mount 2.2.1.1 Elastomeric Mount 2.2.1.2 Hydraulic Mount 2.2.2 Semi Active and Active Engine Mounts 2.3 Controller 2.4 Neural Network Control 2.5 Summary CHAPTER 3: MODELING OF THE ENGINE MOUNTING SYSTEM 3.1 Introduction	
 2.2.1Passive Engine Mount	
 2.2.1Passive Engine Mount 2.2.1.1 Elastomeric Mount 2.2.1.2 Hydraulic Mount 2.2.2 Semi Active and Active Engine Mounts 2.3 Controller 2.4 Neural Network Control 2.5 Summary CHAPTER 3: MODELING OF THE ENGINE MOUNTING SYSTEM 3.1 Introduction 3.2 Passive Engine Mounting 3.2.1.Single Degree of Freedom	
 2.2.1Passive Engine Mount 2.2.1.1 Elastomeric Mount 2.2.1.2 Hydraulic Mount 2.2.2 Semi Active and Active Engine Mounts 2.3 Controller 2.4 Neural Network Control 2.5 Summary CHAPTER 3: MODELING OF THE ENGINE MOUNTING SYSTEM 3.1 Introduction 3.2 Passive Engine Mounting 3.2.1.Single Degree of Freedom 3.3 Effectiveness of Engine Mount Analysis	
 2.2.1Passive Engine Mount	
 2.2.1Passive Engine Mount	
 2.2.1Passive Engine Mount	11 11 13 15 21 24 27 32 36 36 36
 2.2.1Passive Engine Mount	11 11 13 15 21 24 27 32 36 36 36 36 40 36 36 36
2.2.1Passive Engine Mount 2.2.1.1 Elastomeric Mount 2.2.1.2 Hydraulic Mount 2.2.2 Semi Active and Active Engine Mounts 2.3 Controller 2.4 Neural Network Control 2.5 Summary CHAPTER 3: MODELING OF THE ENGINE MOUNTING SYSTEM 3.1 Introduction 3.2 Passive Engine Mounting 3.2.1.Single Degree of Freedom 3.3 Effectiveness of Engine Mount Analysis 3.4 Active Engine Mounting System 3.4.1 Single degree of freedom model 3.4.2 Two degree of freedom model 3.4.3 Multi degree of freedom model 3.5 Electromagnetic Actuator Dynamics	11 11 13 15 21 24 27 27 27 27 27 27 27 27 32 36 36 40 43 47

CHAPTER 4: ADAPTIVE ARTIFICIAL NEURAL NETWORK	
CONTROLLERS	51
4.1 Introduction	51
4.2 Backpropagation	54
4.3 Winner Takes All Algorithm	56
4.3 Neural Network Controllers	57
4.2.1 Nonlinear Autoregressive Moving Average (NARMA-L2)	55
4.2.2 Extended Minimal Resource Allocating Network (EMRAN)	58
4.6 Training of the Neural Network Based Controllers	63
4.3 Summary	64
CHAPTER 5: SINGLE DEGREE OF FREEDOM ENGINE VIBRATION	()
	63
5.1 Introduction	63
5.2 Active Control of linear SDOF Engine Mounting Model	65
5.2.1 Proportional Integral Derivative (PID) Controller	65
5.2.2 Nonlinear Autoregressive Moving Average – L2 (NARMA-L2 Neural Controller) 69
5.2.3 Extended Minimal Resource Allocating Network (EMRAN)	72
5.3 Summary	75
CHAPTER 6: TWO DEGREE ENGINE VIBRATION MODEL	78
6.1 Introduction	78
6.2 Active control of linear TDOF engine mounting system	79
6.2.1 Proportional Integral Derivative PID Controller	80
6.2.2 Nonlinear Autoregressive Moving Average-L2 (NARMA-L2) Neural Controller	83
6.2.3 Extended Minimal Resource Allocation Network (EMRAN) No	eural
Controller	86
6.3 Active control of nonlinear TDOF engine mounting system	91
6.3.1 Nonlinear Autoregressive Moving Average L2 (NARMA-L2).	91
6.3.2 Extended Minimal Resource Allocation Network (EMRAN)	
6.4 Summary	98
CHAPTER 7: MULTI DEGREE ENGINE VIBRATION MODEL	99
7.1 Introduction	
7.2 Active Control of MDOF Engine Mounting Model	100
7.2.1 Linear Quadratic Regulator (LOR) Controller	100
7.2.2 Nonlinear Autoregressive Moving Average (NARMA-L2)	
Controller	103
7.2.3 Extended minimal resource allocating network (EMRAN)	106
7.3 summary	100
CHAPTER 8. Robustness Analysis	111
8 1 Introduction	111
8.2 NARMA I 2 robustness analysis	117
8.2.1 Mass Variations	117
8.2.2 Stiffness Variations	112
8 2 3 Damping Coefficient Variation	115

8.2.4 Mass and Stiffness Variations	
8.3 EMRAN Robustness Analysis	
8.3.1 Mass Variations	
8.3.2 Stiffness Variations	
8.3.3 Damping coefficient variations	
8.3.4 Mass and Stiffness Variations	
8.4 Summary	
CHAPTER 9: CONCLUSION	
9.1 Conclusion	
9.2 Highlights and Contribution of the Study	
9.3 Future work and recommendations	
BIBLIOGRAPHY	
PUBLICATIONS	
APPENDIX A: VITA	

LIST OF TABLES

<u>Table No.</u>		Page No.
2.1	Critical Analysis of Exiting Controllers for Vibration Suppression	24
5.1	The parameters for the SDOF engine vibration isolation system	64
5.2	Ziegler-Nichols Tuning for the Regulator for a decay ratio of 0.25	66
5.3	The average magnitude transmitted force reduction comparison	75
6.1	The parameters for the active engine vibration isolation system	79
8.1	Nominal and varying parameters for robustness analysis	111
8.2	Performance comparison between NARMA-L2 and EMRAN controlle	ers 122

LIST OF FIGURE

FIGURE NO.

PAGE NO.

2.1 Elastomeric mount (Courtesy of Westar Industries)	11
2.2 Dynamic Stiffness of Rubber Engine Mount (Swanson, 1993)	12
2.3 Hydraulic mount with inertia track and decoupler (Courtesy of www.landsharkoz.com)	13
2.4 Dynamic stiffness characteristic of hydraulic mount (Ushijima <i>et al.</i> , 1988)	14
 2.5 Dynamic stiffness characteristic of hydraulic mount with decoupler (a) Dynamic stiffness magnitude (b) loss angle theory — experiment (Singh, 2000) 	15
2.6 Mechanical model of the adaptive hydraulic engine mount (Swanson, 1993)	16
 7 Dynamic damping characteristic of an adaptive fluid mount (Miller and Ahmadian, 1992) 	17
2. 8 Model of magnethorheological fluid mount (courtesy of Delphi Corp.)	17
2.9 Mechanical model of the active rubber mount (Miller and Ahmadian, 1992)	18
2.10 Mechanical model of the active hydraulic mount (Miller and Ahmadian, 1992)	19
2.11 Dynamic stiffness of the active rubber mount (Swanson, 1993)	20
2.12 Dynamic stiffness of the active hydraulic mount (Swanson, 1993)	20
3.1 Representation of a single degree of freedom base excited system	27
3.2 Single degree of freedom response for various values of ζ	29
3.3 Bode plot of a SDOF response for an increasing value of stiffness, k	29
3.4 A SDOF of a force induced excitation	30

3.5 Response of a single degree of freedom mass excited system	31
3.6 Bode plot of a single degree of freedom response for a decreasing value of stiffness, k	32
3.7 Isolation system model for a single mount (Swanson <i>et al.</i> , 1992)	33
3.8 Schematic diagram of a single degree of freedom vibration system with actuator	36
3.9 Block diagram of the active engine mounting system	37
3.10 The force transmissibility plot of the single degree of freedom activ engine mounting	ve 39
3.11 Schematic diagram of an active two degree of freedom system	40
3.12 Schematic diagram of the multi degree of freedom engine vibration system	43
3.13 Schematic diagram of the electromagnetic actuator	47
3.14 The open loop force response of the nonlinear electromagnetic actuator	49
4.1 Nonlinear Autoregressive Moving Average Neural Network Structure	56
4.2 Plant identification of the neural network (Demuth and Beale, 1992)	57
4.3 Radial Basis Function Neural Network (Sundararajan <i>et al.</i> , 1999)	59
5.1 Active vibration isolation simulation block diagram	63
5.2 Transmitted force variations of the open loop system	65
5.3 The response of the system based on the quarter decau ratio	66
5.4 The transmitted force variations of the PID controlled SDOF system	67
5.5 The transmitted force variations of the PID controlled SDOF system at the resonance frequency	67

5.6 The transmitted force variations of the PID controlled SDOF system at final periods of the simulation	68
5.7 The transmitted force variations of the NARMA-L2 controlled SDOF system	70
5.8 The transmitted force variations of the NARMA-L2 controlled SDOF system at the resonance frequency	70
5.9 The transmitted force variations of the NARMA-L2 controlled SDOF system at peridos of the simulation	71
5.10 The transmitted force variations of the EMRAN controlled SDOF system	72
5.11 The transmitted force variations of the EMRAN controlled SDOF system at the resonance frequency	73
5.12 The transmitted force variations of the EMRAN controlled SDOF system at final frequency level seconds of the simulation	73
5.13 Frequency response of the active SDOF system using NARMA-L2 and EMRAN neural controllers.	75
5.14 Vibration isolation performance comparison for a linear SDOF (a) PID controller, (b) NARMA-L2 controller and, (c) EMRAN	77
6.1 The transmitted force variations at (a) front mount (b) rear mount	80
6.2 The transmitted force variations of the PID controlled linear TDOF system (a) front mount (b) rear mount	81
6.3 The transmitted force variations of the PID controlled linear TDOF system at the resonance frequency (a) front mount (b) rear mount	82
6.4 The transmitted force variations of the PID controlled linear TDOF system at the final frequency level (a) front mount (b) rear mount	82
6.5 The transmitted force variations of the NARMA-L2 controlled linear TDOF system (a) front mount (b) rear mount	84
6.6 The transmitted force variations of the NARMA-L2 controlled linear TDOF system at the resonance frequency (a) front mount (b) rear mount	84

6.7 The transmitted force variations of the PID controlled linear TDOF system at the final frequency level (a) front mount (b) rear mount	85
6.8 The transmitted force variations of the EMRAN controlled linear TDOF system (a) front mount (b) rear mount	87
6.9 The transmitted force variations of the EMRAN controlled linear TDOF system at the resonance frequency (a) front mount (b) rear mount	88
6.10 The transmitted force variations of the EMRAN controlled linear TDOF system at the final frequency level (a) front mount (b) rear mount	88
6.11 Frequency response of the active SDOF system using NARMA-L2 and EMRAN neural controllers.	89
6.12 Vibration suppression performance comparison for a linear TDOF (a) PID, (b) NARMA-L2 and, (c) EMRAN	90
6.13 Block diagram of the active engine mounting system with nonlinear dynamics	91
6.14 The transmitted force variations of the NARMA-L2 controlled nonlinear TDOF system (a) front mount (b) rear mount	92
6.15 The transmitted force variations of the NARMA-L2 controlled nonlinear TDOF system between 1 second and 10 seconds (a) front mount (b) rear mount	93
6.16 The transmitted force variations of the NARMA controlled nonlinear TDOF system at the resonance frequency(a) front mount (b) rear mount	93
6.17 The transmitted force variations of the NARMA controlled nonlinear TDOF system at the final frequency level(a) front mount (b) rear mount	94
6.18 The transmitted force variations of the EMRAN controlled nonlinear TDOF system (a) front mount (b) rear mount	96
6.19 The transmitted force variations of the ERMAN controlled nonlinear TDOF system at the resonance frequency(a) front mount (b) rear mount	96

6.20 The transmitted force variations of the EMRAN controlled nonlinear TDOF system at the final frequency level(a) front mount (b) rear mount	97
7.1 The transmitted force variations of the chassis for the open loop linear multi degree of freedom (MDOF) system	100
7.2 Active vibation isolatio system usin LQR optimal control	101
7.3 The transmitted force variations of the chassis for the LQR controlled linear multi degree of freedom (MDOF) system	102
7.4 The transmitted force variations of the chassis for the LQR controlled linear multi degree of freedom (MDOF) system at the resonance frequency	102
7.5 The transmitted force variations of the chassis for the LQR controlled linear multi degree of freedom (MDOF) system above the resonance frequency	103
7.6 The transmitted force variations of the chassis for the NARMA-L2 controlled linear multi degree of freedom (MDOF) system	104
7.7 The transmitted force variations of the chassis for the NARMA-L2 controlled linear multi degree of freedom (MDOF) system at the resonance frequency	105
7.8 The transmitted force variations of the chassis for the NARMA-L2 controlled linear multi degree of freedom (MDOF) system above the resonance frequency	105
7.9 The transmitted force variations of the chassis for the EMRAN controlled linear multi degree of freedom (MDOF) system	106
7.10 The transmitted force variations of the chassis for the EMRAN controlled linear multi degree of freedom (MDOF) system at the resonance frequency	107
7.11 The transmitted force variations of the chassis for the EMRAN controlled linear multi degree of freedom (MDOF) system above the resonance frequency	108
7.11 Frequency response of the active MDOF system using LQR, NARMA-12 and EMRAN controllers.	108
7.13 Vibration suppression performance comparison for a linear MDOF (a) PID, (b) NARMA-L2 and, (c) EMRAN	110

8.1 Magnitude variations of the NARMA-L2 controlled system when subjected to a step response reference with varying mass	113
8.2 Magnitude variations of the NARMA-L2 controlled system when subjected to a step response reference with varying stiffness	114
8.3 Magnitude variations of the NARMA-L2 controlled system when subjected to a step response reference with varying damping ratio	115
8.4 Magnitude variations of the NARMA-L2 controlled system when subjected to a step response reference with varying mass and stiffness	116
8.5 Magnitude variations of the EMRAN controlled system when subjected to a step response reference with varying mass	117
8.6 Magnitude variations of the EMRAN controlled system when subjected to a step response reference with varying stiffness	118
8.7 Magnitude variations of the EMRAN controlled system when subjected to a step response reference with varying damping coefficient	119
8.8 Magnitude variations of the EMRAN controlled system when subjected to a step response reference with varying mass and stiffness	120

\

LIST OF ABBREVIATIONS AND SYMBOLS

PID	Proportional Intergral Derivative
LQR	Linear Quadratic Regulator
NARMA-L2	Nonlinear Autoregressive Moving Average – L2
EMRAN	Extended Minimal Resource Allocating Network
SDOF	Single Degree of Freedom
TDOF	Two Degree of Freedom
MDOF	Multi Degree of Freedom
MR	Magneto Rheological
ER	Electro Rheological
LMS	Least mean square
MIMO	Multi Input Multi Output
EKF	Extended Kalman Filter
ANN	Artificial Neural Network
F	Force
Μ	Moment
m	Mass
k	Stiffness
С	Damping Coefficient
x	Displacement
heta	Angular Displacment
ω	Natural Frequency
ζ	Damping Ratio

CHAPTER 1 INTRODUCTION

3.1 OVERVIEW

Vehicle weight reduction has been a major topic in the automotive industry since it leads to better fuel consumption and better efficiency. Furthermore, with the current environment situation more and more car manufacturers are looking for alternative hydrocarbon fuels to reduce pollution. However, it is known that alternative power trains such as hybrid engines produce less power compared to the traditional internal combustion engines. Looking at the aspect of power to weight ratio alternative power trains could somehow have an equal performance if not better than internal combustion engine provided the weight of the vehicle can be reduced up to an acceptable level.

However, with this trend of lighter vehicle and more powerful engine has led to an undesirable effect to the comfort of the passenger. This undesirable effect has increased the level of noise, vibration and harshness (NVH) to the vehicle especially at the idling frequency of the engine. Since the engine disturbance is directly transmitted through the engine mounts therefore a lot of effort has been focused in improving engine mount technology (Yu *et al.*, 2001). Engine mounting is one of the essential components in the automobile to basically support the weight of the engine. However, despite its simple design the engine mountings have other complex functions.

1.1.1 Passive Engine Mounts

It was reported by Yu et al. (2001) that the passive engine mountings have three purposes. The main purpose is to support the weight of the engine, the second purpose is to isolate the vibration induced by the engine to the chassis and lastly to prevent the engine from bouncing off the chassis would be the third purpose. It was reported by Swanson (1993) and Yang (2001) that engine induced disturbance occurred at frequency between 20 Hz to 200 Hz. This disturbance is mostly caused by the dynamics of the engine components such as pistons, connecting rods and crank shaft as well as the firing pulse (Swanson, 1993; Geisberger, 2000; Krysinski and Malburet, 2007). At this frequency range level for an ideal engine mount to isolate the disturbance effectively the stiffness and damping ratio would be required to be as small as possible. However, at the lower frequency level i.e. below 20 Hz engine is subjected to bounce due to road excitation. To prevent any damages, the stiffness and damping ratio of the engine mount are required to be as large as possible to minimize the relative displacement between the engine and the chassis. This has led to contradictory desirable characteristics of the passive engine mount at both lower frequency and higher frequency levels respectively.

1.1.2 Semi active engine mount

Semi active engine mounting system consists of smart fluids such as electrorheological (ER) fluid or magnetorheological (MR) fluid. The fluids function as adaptive damper that can change their dynamic damping characteristic by applying electric field for ER fluid and by applying magnetic field for MR fluid. Semi active engine mounting systems are normally implemented in a open-loop control architecture. However, these systems are sensitive to the changes in system parameters which make them less robust and they are mostly implemented at the lower frequency range. For the improvement at the higher frequency range a fully active system is implemented.

1.1.3 Active engine mount

To improve the trade-off characteristic of the passive engine mount one alternative is to introduce an active engine mounting system. Active engine mounting system consists of passive mounts such as rubber or hydraulic, an external force actuator and a control system. Different types of force actuators such as electromagnetic, servohydraulic, electrostrictive and magnetostrictive materials could be incorporated into the system. With regards to the control system feedforward or feedback type are commonly used. Although there a lot of controllers which have been created or designed for this purpose, most of the controllers found in the literature are either classical or modern controllers.

Due to the complexity of the system an advance controller such as the neural network has been designed and implemented rather than the classical or modern control, which do not work well for nonlinear system. Furthermore, classical or modern control requires an accurate model to identify the desired controller parameters which is more often than not time consuming and complex. With the ability to be trained on line it was expected that the neural controller would be more robust compared to the classical controller.

3.2 PROBLEM STATEMENT AND ITS SIGNIFICANCE

Cars are becoming an integral part of our daily lives where in most areas they are the major mode of transportation. Engine as the heart of any vehicles but at the same time, engines are acting as a vibration exciter and the need to eliminate or minimize this vibration is essential in which active engine mounts are the solution to this problem.

The vibration sources in automotive are many and one of the major sources is the engine. To have a more comfortable vehicle, these vibrations need to be reduced or ideally eliminated. In this work the reduction of vibration is done through the implementation of the active engine mounting system.

3.3 RESEARCH PHILOSOPHY

With the trend of the numerous applications of intelligent control the philosophy of this research was to identify the possibility of the neural network controller as disturbance rejection in the automotive application namely the active engine mounting system. With its capability to be trained without having a prior mathematical model of the system neural network controllers make a good candidate as robust and practical control architecture. Furthermore, neural network controllers are relatively new especially in the automotive industries which provide broad implementation possibilities.

3.4 RESEARCH OBJECTIVES

The objectives of this research are to:-

1. Develop mathematical models for engine mounting system.

- 2. To benchmark the neural network based controller results against classical PID controllers for a SDOF and TDOF models.
- To investigate the performance of LQR and neural network based controllers to actively isolate the vibration induced by the engine to the chassis for a MDOF model.
- 4. To compare the results obtained between NARMA-L2 and EMRAN controllers for the purpose of engine vibration isolation.
- 5. To investigate the robustness of the neural network based controllers.

3.5 SCOPE

This research is mainly focusing on the simulation of the active engine mounting system. Two types of neural network controllers are implemented in the simulation of the engine vibration system which are the Nonlinear Autoregressive Moving Average L2 (NARMA–L2) and the Extended Minimal Resource Allocating Network (EMRAN). NARMA-L2 has been identified by Narendra (1996), Narendra and Mukhopadhyay (1997) and it has the capability of being trained offline and be used as a controller to reject disturbances, while the EMRAN can be trained online, thus making EMRAN a more robust intelligent controller.

The simulation results of both controllers are then compared with classical PID controller and a Linear Quadratic Regulator controller.